

## Newest capabilities of DIESEL-RK ( 2017 )

**1. Parallel simulation of engine working processes (turbocharging, gasexchange, mixture formation, combustion, emissions formation) and Finite Element Analysis (FEA) of temperature state of engine parts.** FEA analysis has advanced interface: user can select design of cylinder liner, head and piston from data bases, combine them using easy drag 'n' drop technique and assign materials of main parts in assembly (see figures below).

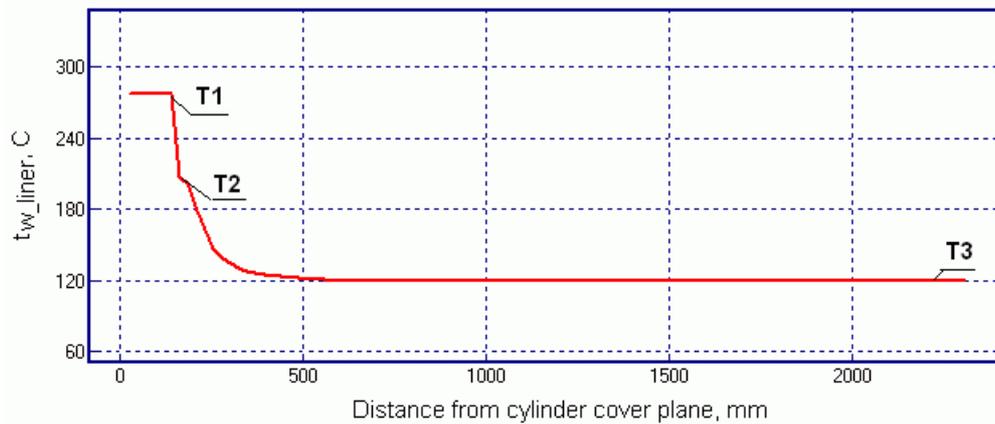
The screenshot displays the 'Cylinder-Piston Assembly' software interface. The main window shows a 3D model of the engine components with various heat transfer parameters labeled. The parameters include  $h_{c4}$ ,  $T_{c4}$ ,  $h_{c3}$ ,  $T_{c3}$ ,  $h_{c2}$ ,  $T_{c2}$ ,  $h_{c1}$ ,  $T_{c1}$ ,  $h_{c0}$ ,  $T_{c0}$ ,  $h_{p1}$ ,  $T_{p1}$ ,  $h_{p3}$ ,  $T_{p3}$ ,  $h_{p2}$ , and  $T_{p2}$ . The graph shows the Normalized Heat Transfer Coefficient Profile across 10 nodes. The y-axis is 'Heat Transfer Coefficient Profile' (ranging from 10 to 40) and the x-axis is 'Node number (see picture)' (ranging from 1 to 10). The profile shows a peak of approximately 45 at node 7. The software also includes a database of engine designs and material selection options.

Node number	Heat Transfer Coefficient Profile
1	10
2	10
3	15
4	20
5	35
6	40
7	45
8	40
9	35
10	30

**Materials**

Material of Liner	Bohler W720 m
Material of Head (Part #1)	High Test Iron
Material of Gasket	M
Material of (Part #2)	42CrMo4V

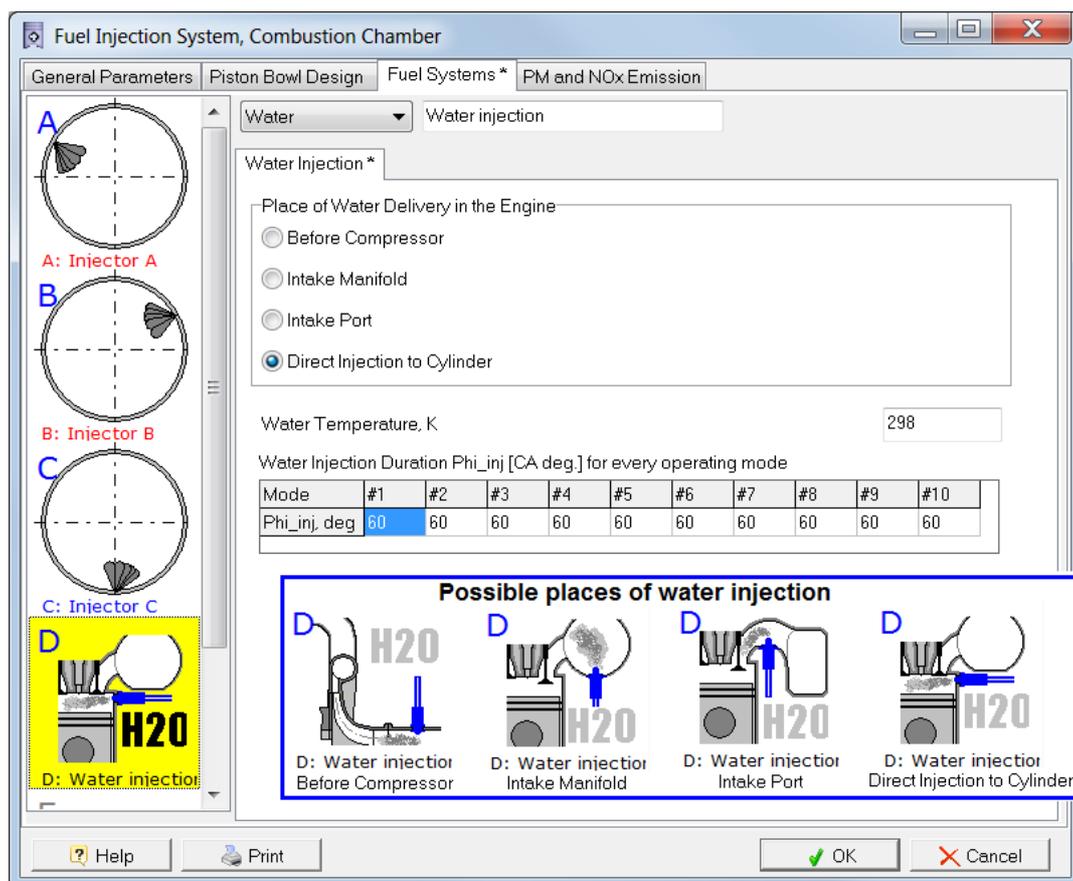
This function allows accurate simulation of heat exchange during intake and scavenging because allows prediction temperature field of long cylinder liner and place where piston is in TDC. The typical approximation of the Cylinder Liner Temperature obtained for large marine two-stroke engine is presented below.



Correct temperature field of engine parts allows right simulation of gas flow at gasexchange and evaporation in the zones where the fuel spray impinges the wall. Program will scale the assembly for actual engine D/S, implement the actual piston bowl into piston crown, make a grid and will perform simulation – everything automatically.

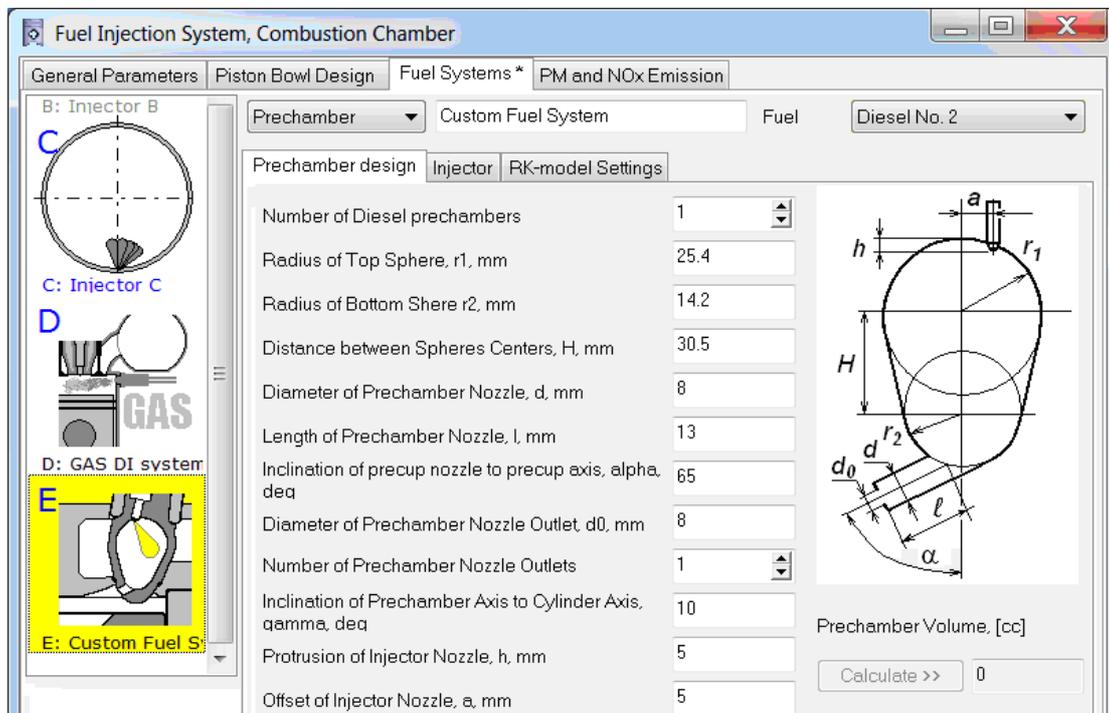
In result, together with engine cycle parameters there temperature fields and critical temperatures of piston rings and all surfaces are outputted.

**2. Simulation of engine with few (up to 5) independent fuel injection systems including Gas and Water injection system.** Injection systems are marked by indexes A, B, C, D and E. Any system may be intended for delivery Liquid fuel, Gas or Water. Gas and Water may be injected into different places. Condensation of water from ambient air is accounted as well.

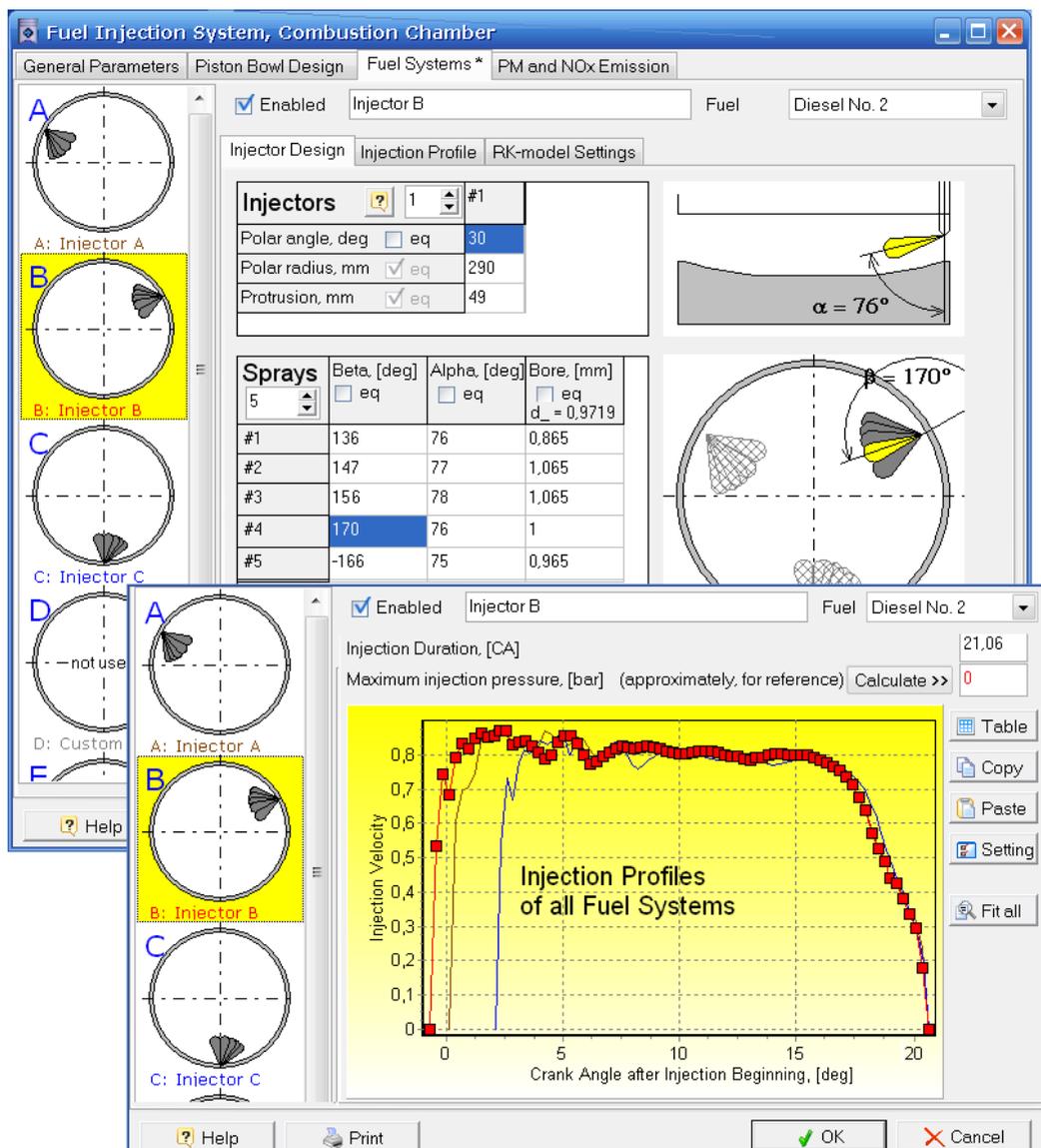


In the Gas engine with ignition by diesel oil the delivery of Gas is possible to be simulated in: Intake Manifold; Intake Port; before Compressor and Directly into Cylinder, as injection of water in above picture. Ignition of Gas is possible to be simulated as by DI of diesel oil, as by diesel injection into Pre-chamber.

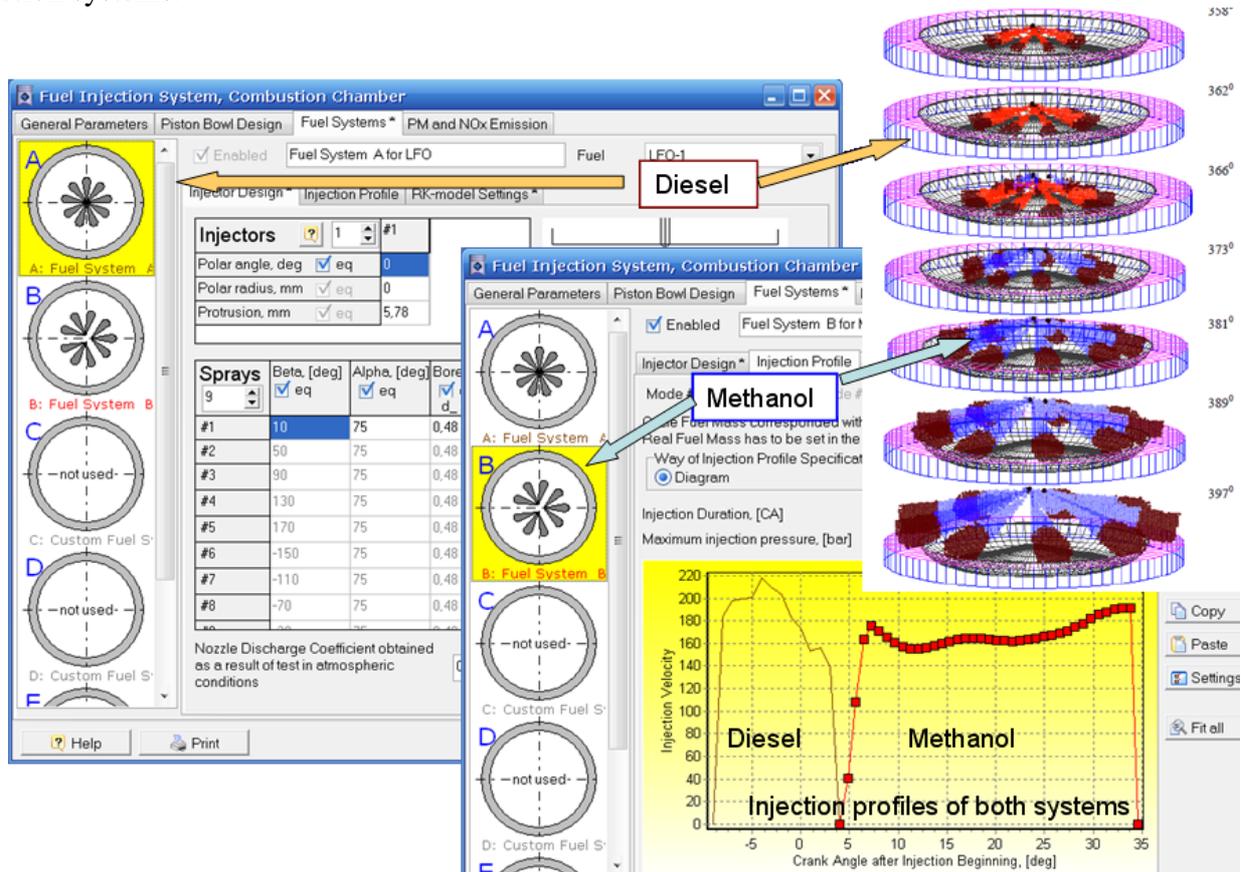




If the engine has few DI diesel system it is possible to set own design and own injection profile for every system.

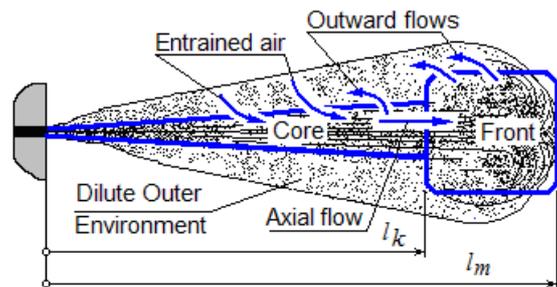
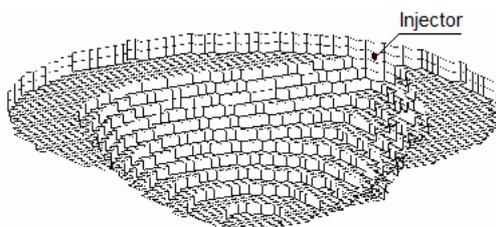


**3. Simulation of Dual Fuel engine.** This feature is realized if different fuels are specified for fuel injection systems.

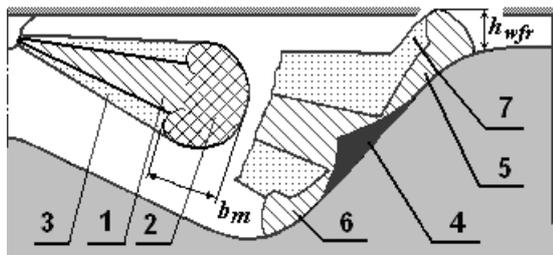


**4. Simulation of spatial fuel sprays intersection.** 3D mesh is used for sprays shape interpretation and piston bowl shape interpretation. Gas flows inside and around of the sprays are simulated using equation of momentum conservation. It is non-CFD simulation: balance equations are resolved for clusters of cells but not in each cell. Computational time is 1 ... 2 min on conventional PC.

In cylinder 3D mesh of OP engine at TDC



Sprays are divided into 11 characteristic zones.



1. The dense conical core of free spray;
2. The dense forward front of free spray;
3. The dilute outer sleeve of free spray.
4. The axial conical core of the Near Wall Flow;
5. The dense core of the Near Wall Flow on the piston bowl surface;
6. The dense forward front of the Near Wall Flow;
7. The dilute outer zone of the Near Wall Flow;

8. Zone on the cylinder liner surface if fuel reaches it.

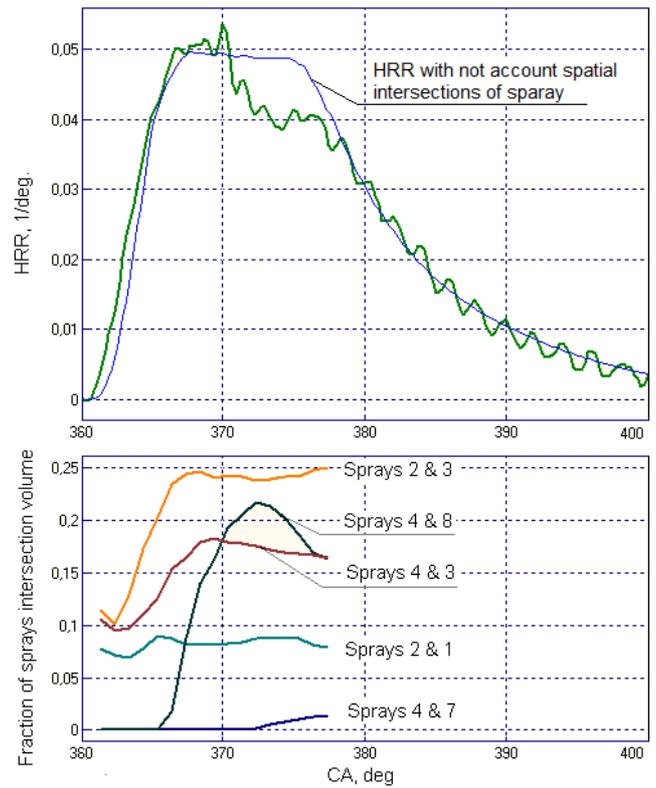
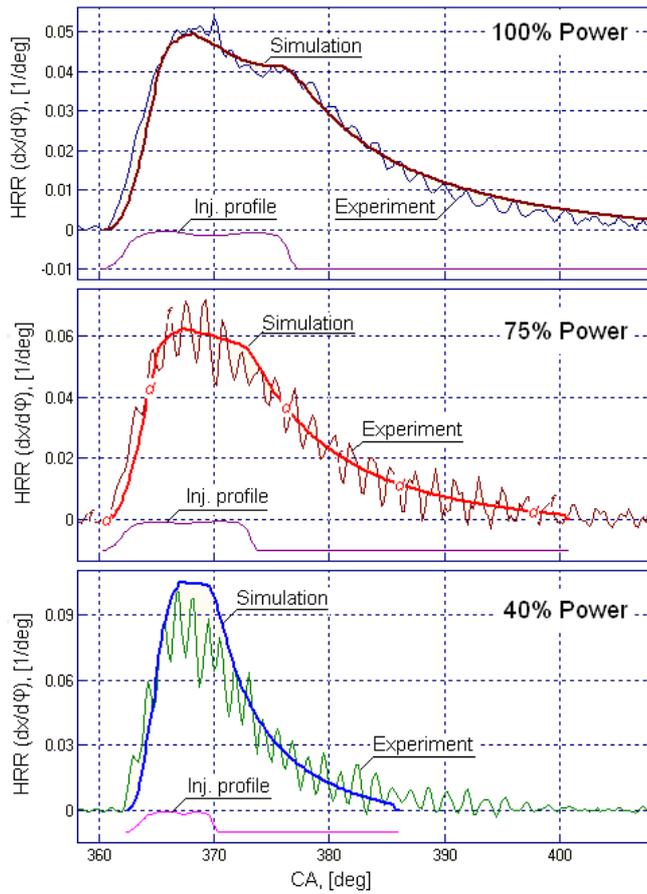
9. Zone on the cylinder head if fuel reaches it.

10. Zone of Near Wall Flow overlap.

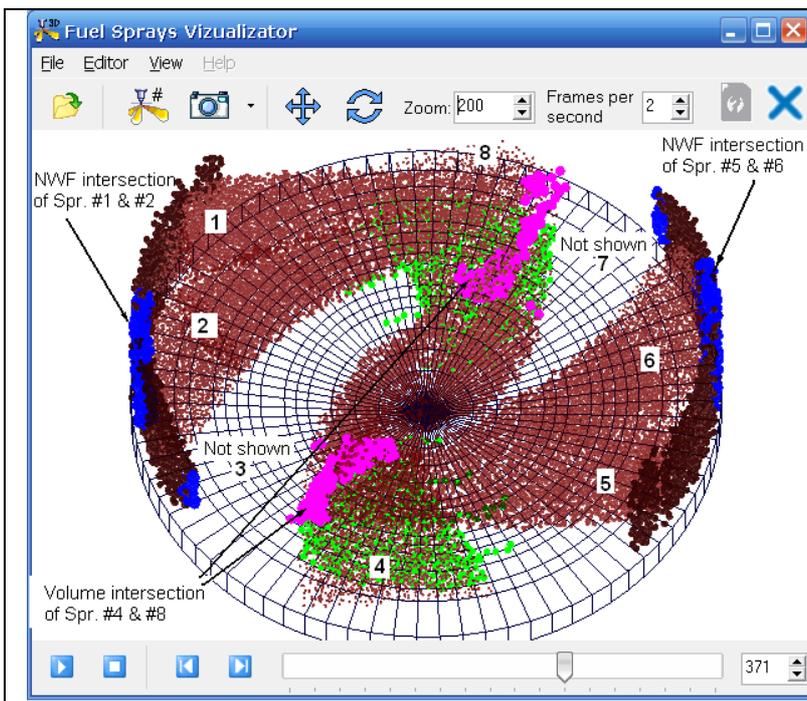
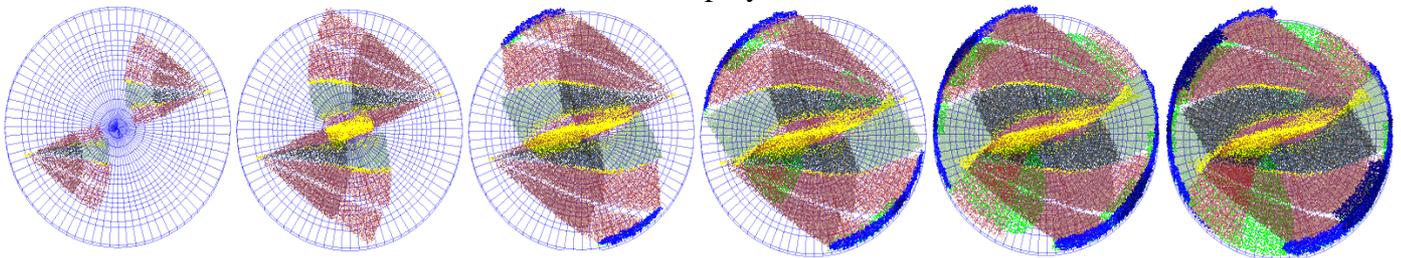
11. Zone of sprays spatial overlap.

Each zone has own character temperatures dependent on time and own conditions of evaporation.

Account of spatial sprays intersection allows simulation of character decrease of heat release rate after middle of injection at large load.



The diagrams show most massive overlap of sprays



Evolution of sprays from 2 injectors in cylinder of 2 stroke large marine engine with side injection system. (One inner spray is marked by green bullets to separate inner and outer sprays. Light Green bullets are Near Wall Flow (NWF) on piston; Blue bullets are NWF on cylinder wall. Dark Blue bullets are NWF overlap.) Yellow bullets on figure indicate zones of sprays spatial intersections. The intersections lead to local decrease of HRR.

Program for 3D Fuel Spray Visualization allows marking of sprays and their zones to analyze sprays overlap and search for solutions to minimize the overlapping.